



*Mathematical model for a satellite  
with hinged flexible elements*

Stepan Tkachev

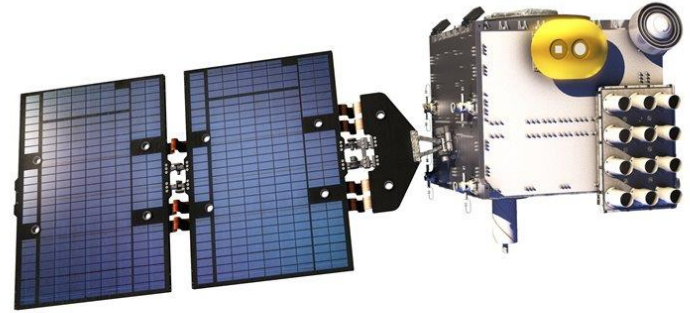
Keldysh Institute of Applied Mathematics

# Main topics

- Introduction
- Problem definition
- Satellite with antenna and hinged solar panel
- Satellite with hinged solar panel
- Numerical modeling

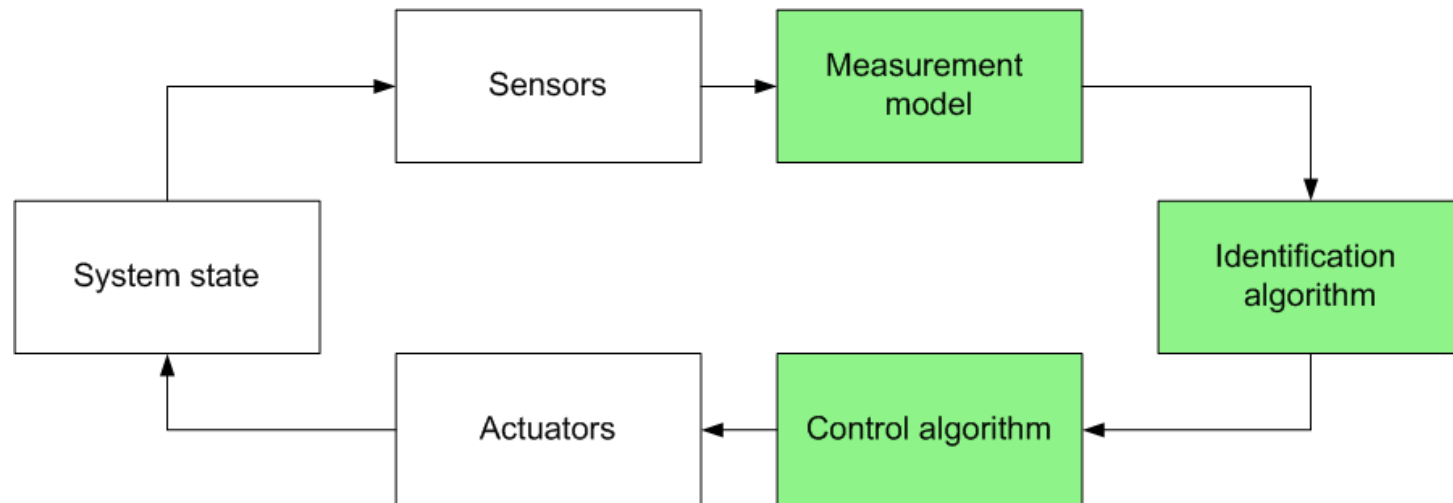
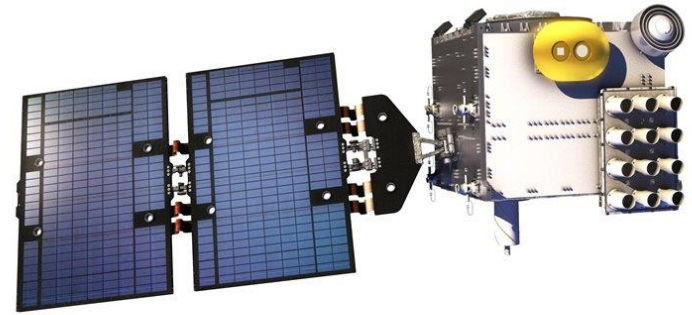
# Angular motion models

- Solar panels
- Antennas
- Scientific equipment
- Ground precise model  
usually non-linear
- On-board simplified model  
usually linear



# Angular motion models

- Solar panels
- Antennas
- Scientific equipment
- Ground precise model
- On-board simplified model



# Equations derivation approaches

Satellite with flexible elements has almost infinite degrees of freedom

Usage of partial differential equations

Pros

- Precise
- Rather simple form

Cons

- Difficult to solve numerically
- Difficult to use for algorithms synthesis

Usage of normal modes

Pros

- Easy to analyze
- Simple to solve numerically

Cons

- Precision varies

# Equations derivation approaches

Satellite with flexible elements has almost infinite degrees of freedom

Usage of partial differential equations

Pros

- Precise
- Rather simple form

Cons

- Difficult to solve numerically
- Difficult to use for algorithms synthesis

Usage of normal modes

Pros

- Easy to analyze
- Simple to solve numerically

Cons

- Precision varies

Modes with low frequencies and high amplitudes are left

- The most part of oscillation energy
- Slow damping

# Normal (Eigen) modes

Degrees of freedom

3 for center of mass motion

3 for angular motion

$N$  for flexible deformation (normal modes)

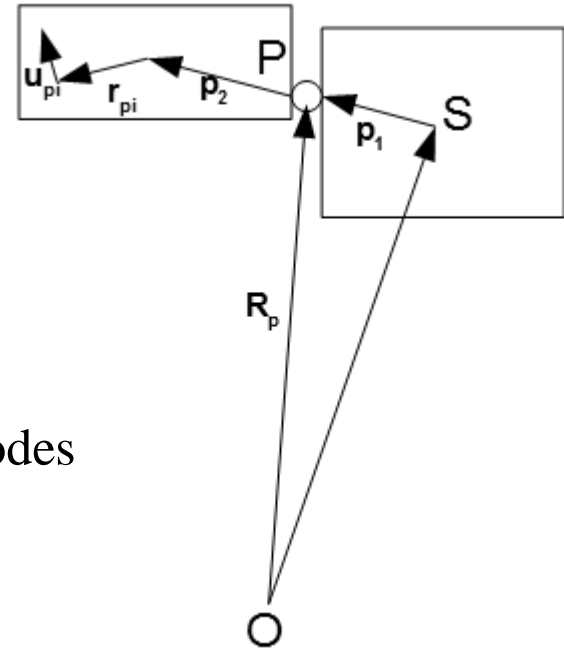
Deformations

$$\mathbf{u}_{pi} = \mathbf{A}_{pi}(\mathbf{r}_{pi})\mathbf{q}(t), \quad \mathbf{A}_{pi}(\mathbf{r}_{pi}) - 3 \times N - \text{forms of modes}$$

$$\mathbf{q}(t) \quad - \text{amplitudes}$$

Langrangian equations or general dynamics  
equation can be used

$$\sum_i (m_i \ddot{\mathbf{R}}_i - \mathbf{F}_i) \delta \mathbf{R}_i = 0$$



# Problem definition

Several flexible parts  
At least one of the rotating w.r.t. to the bus



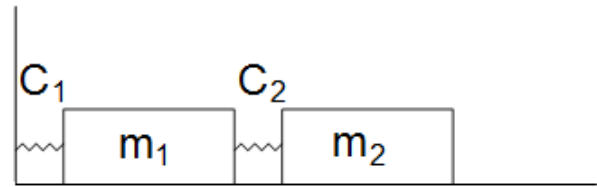
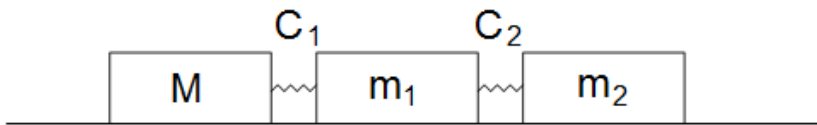
Normal modes of the whole satellite are varying



Using the normal modes for each flexible element



# Small dimension example



$$L_1 = \frac{1}{2}M\dot{x}^2 + \frac{1}{2}m_1(\dot{x} + \dot{y}_1)^2 + \frac{1}{2}m_2(\dot{x} + \dot{y}_1 + \dot{y}_2)^2 - \frac{1}{2}C_1y_1^2 - \frac{1}{2}C_2y_2^2$$

$$L_1 = \frac{1}{2}(\dot{x} \quad \dot{\mathbf{y}})\mathbf{A}_1\begin{pmatrix} \dot{x} \\ \dot{\mathbf{y}} \end{pmatrix} + \frac{1}{2}\mathbf{y}^T\mathbf{C}\mathbf{y}$$

$$L_1 = \frac{1}{2}(M + m_1 + m_2)\dot{x}^2 + \dot{x}(m_1 \quad m_2)\mathbf{U}\dot{\boldsymbol{\theta}} + \frac{1}{2}\dot{\boldsymbol{\theta}}^2 - \frac{1}{2}\boldsymbol{\theta}^T\boldsymbol{\Lambda}\boldsymbol{\theta}$$

$$L_2 = \frac{1}{2}m_1\dot{y}_1^2 + \frac{1}{2}m_2(\dot{y}_1 + \dot{y}_2)^2 - \frac{1}{2}C_1y_1^2 - \frac{1}{2}C_2y_2^2$$

$$L_2 = \frac{1}{2}\dot{\mathbf{y}}^T\mathbf{A}_2\dot{\mathbf{y}} + \frac{1}{2}\mathbf{y}^T\mathbf{C}\mathbf{y}$$

$$L_2 = \frac{1}{2}\dot{\boldsymbol{\theta}}^2 - \frac{1}{2}\boldsymbol{\theta}^T\boldsymbol{\Lambda}\boldsymbol{\theta}, \quad \mathbf{y} = \mathbf{U}\boldsymbol{\theta}$$

$\boldsymbol{\theta}$  – normal coordinates,  $\mathbf{I}_{2 \times 2} = \mathbf{U}^T\mathbf{A}_2\mathbf{U}$ ,  $\boldsymbol{\Lambda} = \mathbf{U}^T\mathbf{C}\mathbf{U}$

$$\mathbf{A}_1 = \begin{pmatrix} M + m_1 + m_2 & (m_1 \quad m_2)\mathbf{U} \\ \mathbf{U}^T(m_1 \quad m_2)^T & \mathbf{I}_{2 \times 2} \end{pmatrix}$$

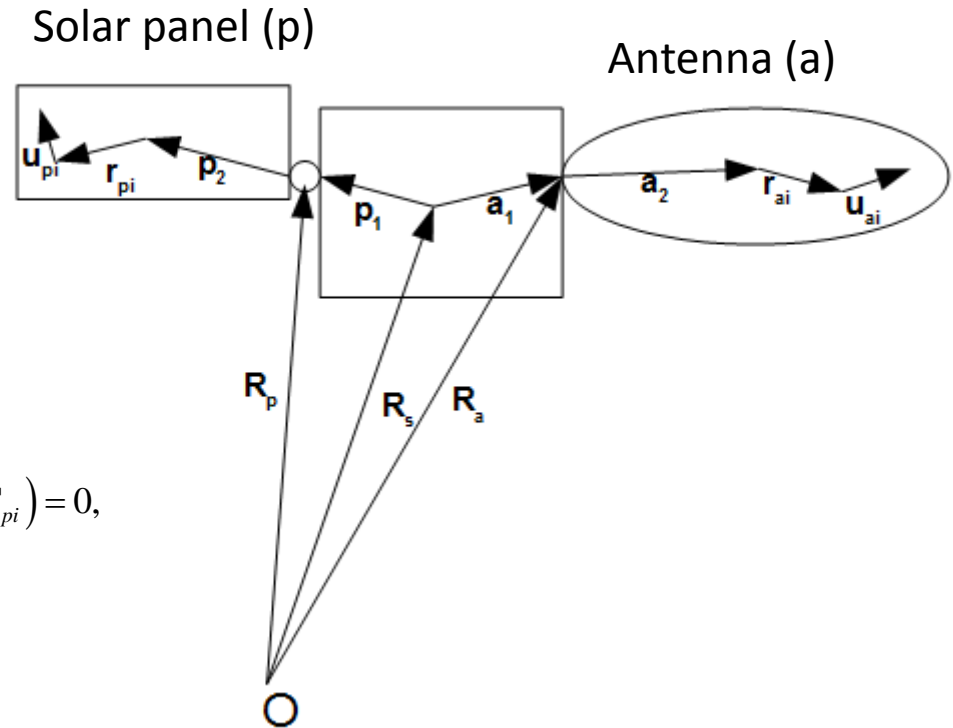
# 3 part satellite

## Satellite

Rigid bus

Fixed flexible antenna

Hinged flexible solar panel



$$\sum_i (m_{si} \ddot{\mathbf{R}}_{si} - \mathbf{F}_{si}) + \sum_i (m_{ai} \ddot{\mathbf{R}}_{ai} - \mathbf{F}_{ai}) + \sum_i (m_{pi} \ddot{\mathbf{R}}_{pi} - \mathbf{F}_{pi}) = 0,$$

$$\sum_i \mathbf{e}^T \left( (\mathbf{p}_2 + \mathbf{r}_{pi} + \mathbf{u}_{pi}) \times (m_{pi} \ddot{\mathbf{R}}_{pi} - \mathbf{F}_{pi}) \right) = M_e,$$

$$\sum_i \mathbf{A}_{ai}^T (m_{ai} \ddot{\mathbf{R}}_{ai} - \mathbf{F}_{ai}) = 0,$$

$$\sum_i \mathbf{A}_{pi}^T (m_{pi} \ddot{\mathbf{R}}_{pi} - \mathbf{F}_{pi}) = 0,$$

$$\sum_i \mathbf{r}_{si} \times (m_{si} \ddot{\mathbf{R}}_{si} - \mathbf{F}_{si}) + \sum_i (\mathbf{a}_1 + \mathbf{r}_{ai} + \mathbf{u}_{ai}) \times (m_{ai} \ddot{\mathbf{R}}_{ai} - \mathbf{F}_{ai}) + \sum_i (\mathbf{p}_1 + \mathbf{r}_{pi} + \mathbf{u}_{pi}) \times (m_{pi} \ddot{\mathbf{R}}_{pi} - \mathbf{F}_{pi}) = 0.$$

+ kinematic equations

# 3 part satellite

$$\begin{aligned}
 & \mathbf{J}\dot{\boldsymbol{\omega}} + \mathbf{S}_{\omega\varphi} \mathbf{e}\dot{\psi} + \mathbf{S}_{\omega a} \ddot{\mathbf{q}}_a + \mathbf{S}_{\omega p} \ddot{\mathbf{q}}_p + \boldsymbol{\omega} \times \mathbf{J}\boldsymbol{\omega} + 2m_a \mathbf{a}_1 \times \boldsymbol{\omega} \times \ddot{\mathbf{a}}_2 + 2 \sum_i (\mathbf{r}_{ai} + \mathbf{u}_{ai}) \times m_{ai} \boldsymbol{\omega} \times \dot{\mathbf{u}}_{ai} + 2 \sum_i (\mathbf{r}_{pi} + \mathbf{u}_{pi}) \times m_{pi} \boldsymbol{\omega}_2 \times \dot{\mathbf{u}}_{pi} + \\
 & + \tilde{\mathbf{J}}_p (\boldsymbol{\omega} \times \mathbf{e}\psi) + \boldsymbol{\omega} \times \tilde{\mathbf{J}}_p \mathbf{e}\psi + \mathbf{e}\psi \times \tilde{\mathbf{J}}_p \boldsymbol{\omega} + \mathbf{e}\psi \times \tilde{\mathbf{J}}_p \mathbf{e}\psi - 2 \frac{1}{m} (m_a \mathbf{a} + m_a \tilde{\mathbf{a}}_2 + m_p \mathbf{p} + m_p \tilde{\mathbf{p}}_2) \times m_a \boldsymbol{\omega} \times \dot{\mathbf{a}}_2 + \\
 & + \left( m_p \mathbf{p}_1 - \frac{1}{m} (m_a \mathbf{a} + m_a \tilde{\mathbf{a}}_2 + m_p \mathbf{p} + m_p \tilde{\mathbf{p}}_2) \right) \times \left( 2m_p \boldsymbol{\omega} \times \mathbf{e}\psi \times (\mathbf{p}_2 + \tilde{\mathbf{p}}_2) + m_p \mathbf{e}\psi \times \mathbf{e}\psi \times (\mathbf{p}_2 + \tilde{\mathbf{p}}_2) + 2m_p \boldsymbol{\omega}_2 \times \dot{\tilde{\mathbf{p}}}_2 \right) + \mathbf{f}_a + \mathbf{f}_p - \mathbf{T}_s = 0,
 \end{aligned}$$

$$\begin{aligned}
 & \mathbf{S}_{\omega p}^T \dot{\boldsymbol{\omega}} + \mathbf{S}_{\varphi p}^T \mathbf{e}\dot{\varphi} + \mathbf{S}_{ap}^T \ddot{\mathbf{q}}_a + \mathbf{M}_p \ddot{\mathbf{q}}_p + \sum_i m_{pi} \mathbf{A}_{pi}^T \boldsymbol{\omega} \times \boldsymbol{\omega} \times \left( \mathbf{r}_{pi} + \mathbf{A}_{pi} \mathbf{q}_p + \mathbf{p}_1 - \frac{1}{m} \left( m_a \mathbf{a} + \sum_i m_{ai} \mathbf{A}_{ai} \mathbf{q}_a + m_p \mathbf{p} + \sum_i m_{pi} \mathbf{A}_{pi} \mathbf{q}_p \right) \right) + \\
 & + \sum_i m_{pi} \mathbf{A}_{pi}^T \mathbf{e}\dot{\varphi} \times \mathbf{e}\dot{\varphi} \times \left( \mathbf{r}_{pi} + \mathbf{A}_{pi} \mathbf{q}_p \right) - \frac{1}{m} \left( m_p \mathbf{p}_2 + \sum_i m_{pi} \mathbf{A}_{pi} \mathbf{q}_p \right) - \frac{1}{m} \mathbf{e}\dot{\varphi} \times \left( m_p \mathbf{p}_2 + \sum_i m_{pi} \mathbf{A}_{pi} \mathbf{q}_p \right) + \\
 & + 2 \sum_i m_{pi} \mathbf{A}_{pi}^T \boldsymbol{\omega} \times \left( \mathbf{A}_{pi} \dot{\mathbf{q}}_p + \mathbf{e}\dot{\varphi} \times (\mathbf{r}_{pi} + \mathbf{A}_{pi} \mathbf{q}_p) \right) - \frac{1}{m} \left( \sum_i m_{ai} \mathbf{A}_{ai} \dot{\mathbf{q}}_a + \sum_i m_{pi} \mathbf{A}_{pi} \dot{\mathbf{q}}_p \right) - 2 \frac{1}{m} \sum_i m_{pi} \mathbf{A}_{pi}^T \mathbf{e}\dot{\varphi} \times \sum_i m_{pi} \mathbf{A}_{pi} \dot{\mathbf{q}}_p + \\
 & + \mathbf{f}_{qp} + \sum_i m_{pi} \mathbf{A}_{pi}^T \mathbf{A}_{pi} (\mathbf{D}\dot{\mathbf{q}}_p + \boldsymbol{\Omega} \mathbf{q}_p) = 0.
 \end{aligned}$$

$$\mathbf{S} \begin{pmatrix} \dot{\boldsymbol{\omega}} \\ \dot{\psi} \\ \ddot{\mathbf{q}}_a \\ \ddot{\mathbf{q}}_p \end{pmatrix} = \mathbf{N}(\boldsymbol{\omega}, \psi, \dot{\mathbf{q}}_a, \dot{\mathbf{q}}_p, \mathbf{A}, \varphi, \mathbf{q}_a, \mathbf{q}_p), \quad \mathbf{S} = \begin{pmatrix} \mathbf{J} & \mathbf{S}_{\omega\varphi} \mathbf{e} & \mathbf{S}_{\omega a} & \mathbf{S}_{\omega p} \\ \mathbf{e}^T \mathbf{S}_{\omega\varphi}^T & \mathbf{e}^T \mathbf{J} \mathbf{e} & \mathbf{e}^T \mathbf{S}_{\varphi a} & \mathbf{e}^T \mathbf{S}_{\varphi p} \\ \mathbf{S}_{\omega a}^T & \mathbf{S}_{\varphi a}^T \mathbf{e} & \mathbf{M}_a & \mathbf{S}_{ap} \\ \mathbf{S}_{\omega p}^T & \mathbf{S}_{\varphi p}^T \mathbf{e} & \mathbf{S}_{ap}^T & \mathbf{M}_p \end{pmatrix}$$

# Linearized equations

Considering

$$\frac{m_a}{m} \ll 1 \quad \frac{m_p}{m} \ll 1$$

$$\mathbf{S} = \begin{pmatrix} \mathbf{J} & \mathbf{S}_{\omega\phi} \mathbf{e} & \mathbf{S}_{\omega a} & \mathbf{S}_{\omega p} \\ \mathbf{e}^T \mathbf{S}_{\omega\phi}^T & \mathbf{e}^T \mathbf{J} \mathbf{e} & 0 & \mathbf{e}^T \mathbf{S}_{\phi p} \\ \mathbf{S}_{\omega a}^T & 0 & \mathbf{M}_a & 0 \\ \mathbf{S}_{\omega p}^T & \mathbf{S}_{\phi p}^T \mathbf{e} & 0 & \mathbf{M}_p \end{pmatrix}$$

$$\mathbf{J}\dot{\boldsymbol{\omega}} + \mathbf{S}_{\omega\phi} \mathbf{e}\ddot{\boldsymbol{\phi}} + \mathbf{S}_{\omega a} \ddot{\mathbf{q}}_a + \mathbf{S}_{\omega p} \ddot{\mathbf{q}}_p + \mathbf{f}_a + \mathbf{f}_p - \mathbf{M}_s = 0,$$

$$\mathbf{e}^T \mathbf{S}_{\omega\phi}^T \dot{\boldsymbol{\omega}} + \mathbf{e}^T \mathbf{J} \mathbf{e} \ddot{\boldsymbol{\phi}} + \mathbf{e}^T \mathbf{S}_{\phi p} \ddot{\mathbf{q}}_p + \mathbf{e}^T \mathbf{f}_p = M_e,$$

$$\mathbf{S}_{\omega a}^T \dot{\boldsymbol{\omega}} + \mathbf{M}_a (\ddot{\mathbf{q}}_a + \mathbf{D}\dot{\mathbf{q}}_a + \boldsymbol{\Omega}\mathbf{q}_a) + \mathbf{f}_{qa} = 0,$$

$$\mathbf{S}_{\omega p}^T \dot{\boldsymbol{\omega}} + \mathbf{S}_{\phi p}^T \mathbf{e}\ddot{\boldsymbol{\phi}} + \mathbf{M}_p (\ddot{\mathbf{q}}_p + \mathbf{D}\dot{\mathbf{q}}_p + \boldsymbol{\Omega}\mathbf{q}_p) + \mathbf{f}_{qp} = 0.$$

$3 + 1 + N_p + N_a$  degrees of freedom

# Satellite with hinged solar panel

$$\mathbf{S} \begin{pmatrix} \dot{\boldsymbol{\omega}} \\ \dot{\boldsymbol{\psi}} \\ \ddot{\mathbf{q}}_p \end{pmatrix} = \mathbf{N}(\boldsymbol{\omega}, \boldsymbol{\psi}, \dot{\mathbf{q}}_p, \mathbf{q}, \boldsymbol{\varphi}, \dot{\mathbf{q}}_p) \quad \mathbf{S} = \begin{pmatrix} \mathbf{J} & \mathbf{S}_{\omega\boldsymbol{\varphi}} \mathbf{e} & \mathbf{S}_{\omega p} \\ \mathbf{e}^T \mathbf{S}_{\omega\boldsymbol{\varphi}}^T & \mathbf{e}^T \mathbf{J} \mathbf{e} & \mathbf{e}^T \mathbf{S}_{\boldsymbol{\varphi} p} \\ \mathbf{S}_{\omega p}^T & \mathbf{S}_{\boldsymbol{\varphi} p}^T \mathbf{e} & \mathbf{M}_p \end{pmatrix}$$

$$\dot{\boldsymbol{\psi}} = 0, \quad \boldsymbol{\varphi} = \text{const}$$

$$\mathbf{J} \dot{\boldsymbol{\omega}} + \mathbf{S}_{\omega p} \ddot{\mathbf{q}}_p = 0,$$

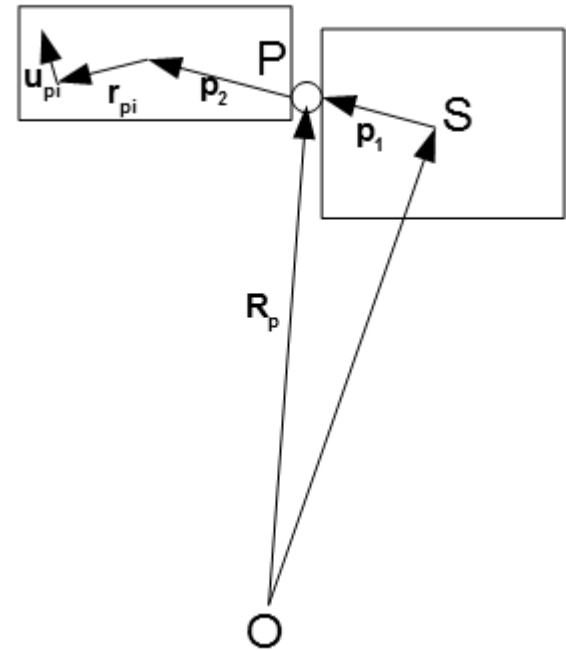
$$\mathbf{S}_{\omega p}^T \dot{\boldsymbol{\omega}} + \mathbf{M}_p \ddot{\mathbf{q}}_p + \sum_i m_{pi} \mathbf{A}_{pi}^T \mathbf{A}_{pi} \boldsymbol{\Omega} \mathbf{q}_p = 0,$$

$$\mathbf{S}_{\omega p} = \sum_i (\mathbf{p}_1 + \mathbf{r}_{pi}) \times m_{pi} \mathbf{A}_{pi} - \frac{1}{m} m_p \mathbf{p} \times \sum_i m_{pi} \mathbf{A}_{pi},$$

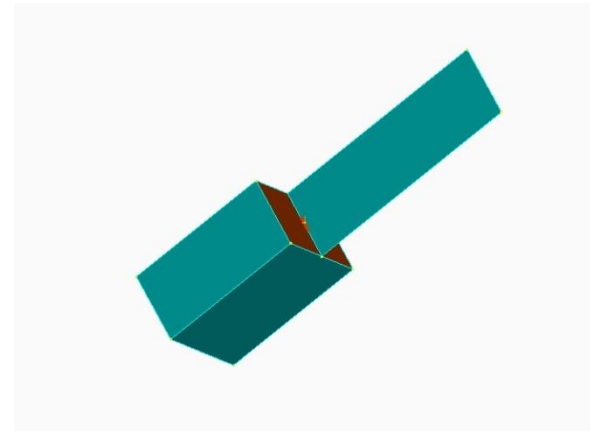
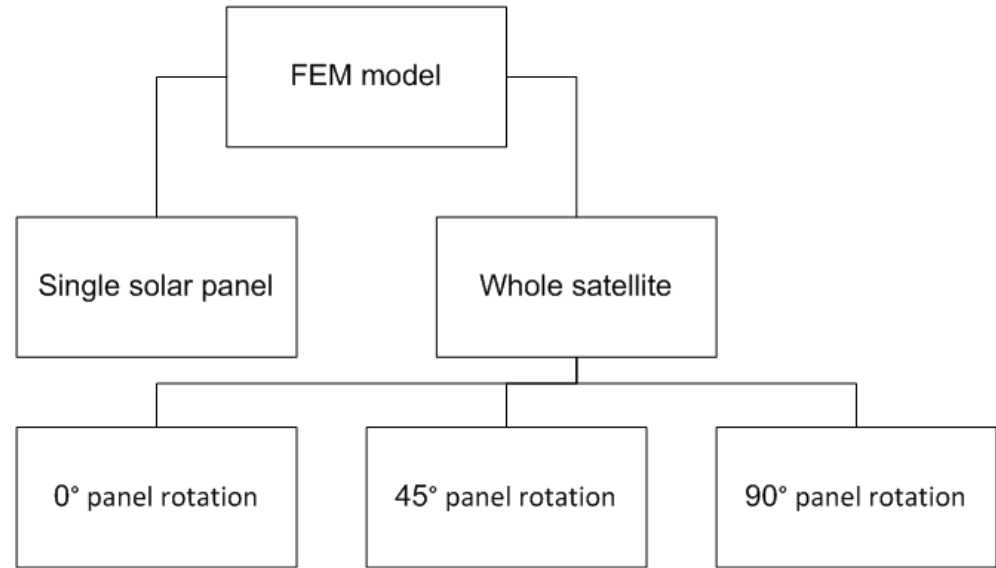
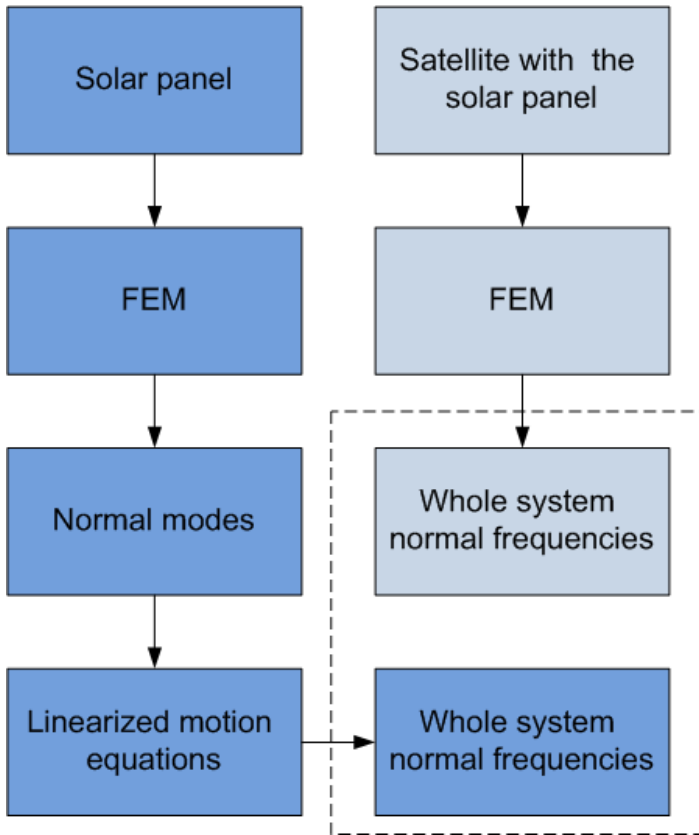
$$\mathbf{J} = \mathbf{J}_s + \tilde{\mathbf{J}}_p + \mathbf{K}(\mathbf{p}_1, m_p \mathbf{p}_1) + \mathbf{K}(\mathbf{p}_1, m_p \mathbf{p}_2) + \mathbf{K}(m_p \mathbf{p}_2, \mathbf{p}_1) - \frac{1}{m} \mathbf{K}(m_p \mathbf{p}, m_p \mathbf{p}),$$

$$\mathbf{M}_p = \sum_i m_{pi} \mathbf{A}_{pi}^T \mathbf{A}_{pi} - \frac{1}{m} \sum_i m_{pi} \mathbf{A}_{pi}^T \sum_i m_{pi} \mathbf{A}_{pi}.$$

$$\mathbf{K}(\mathbf{a}, \mathbf{b}) = \begin{pmatrix} a_2 b_2 + a_3 b_3 & -a_2 b_1 & -a_3 b_1 \\ -a_1 b_2 & a_1 b_1 + a_3 b_3 & -a_3 b_2 \\ -a_1 b_3 & -a_2 b_3 & a_1 b_1 + a_2 b_2 \end{pmatrix}$$



# Numerical modeling



panel grid: 200x600  
satellite grid: 20x20x20

# FEM modeling

## Flexible solar panel

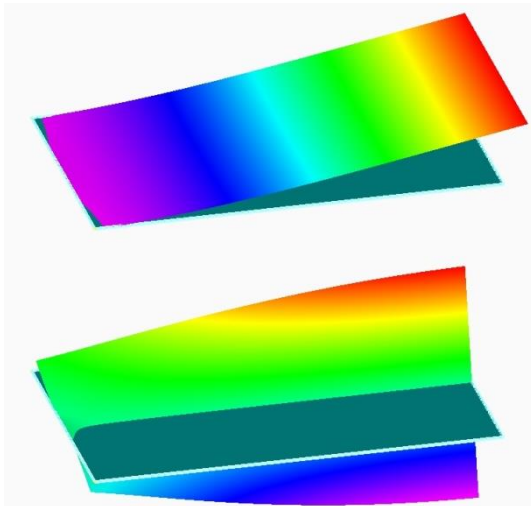
Purposes

### **Main**

Determine quantities for motion equations

### **Auxiliary**

Eigen modes forms and frequencies determination

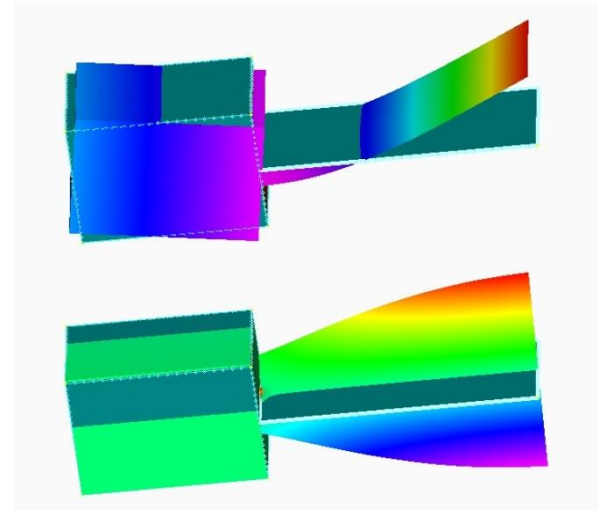


## Satellite with flexible panel

Purpose

### **Auxiliary**

Eigen modes forms and frequencies determination



# Eigen frequencies comparison

Mode No.	Single panel	Satellite with panel $\varphi = 0$		Satellite with panel $\varphi = \pi/4$		Satellite with panel $\varphi = \pi/2$	
		FEM	Model	FEM	Model	FEM	Model
1	5.2487	7.6310	7.6311	7.8223	7.8234	8.0301	8.0316
2	32.057	32.393	32.483	32.378	32.483	32.361	32.483
3	33.256	36.063	36.068	36.195	36.203	36.364	36.370
4	34.485	52.679	53.148	51.235	51.679	49.722	50.137
5	98.195	99.381	99.420	99.488	99.533	99.596	99.643
6	100.90	100.78	101.03	100.73	101.03	100.69	101.02
7	181.44	181.11	181.51	181.04	181.51	180.97	181.51
8	190.81	191.90	192.06	191.98	192.17	192.07	192.28
9	277.50	276.98	277.54	276.89	277.54	276.79	277.54
10	294.77	295.27	295.89	295.26	295.98	295.25	296.08

Model calculation

FEM for panel: 30 minutes

Model auxiliary parameters: 5 minutes

One position mode calculation: less 1 second

Total time: about 35 minutes

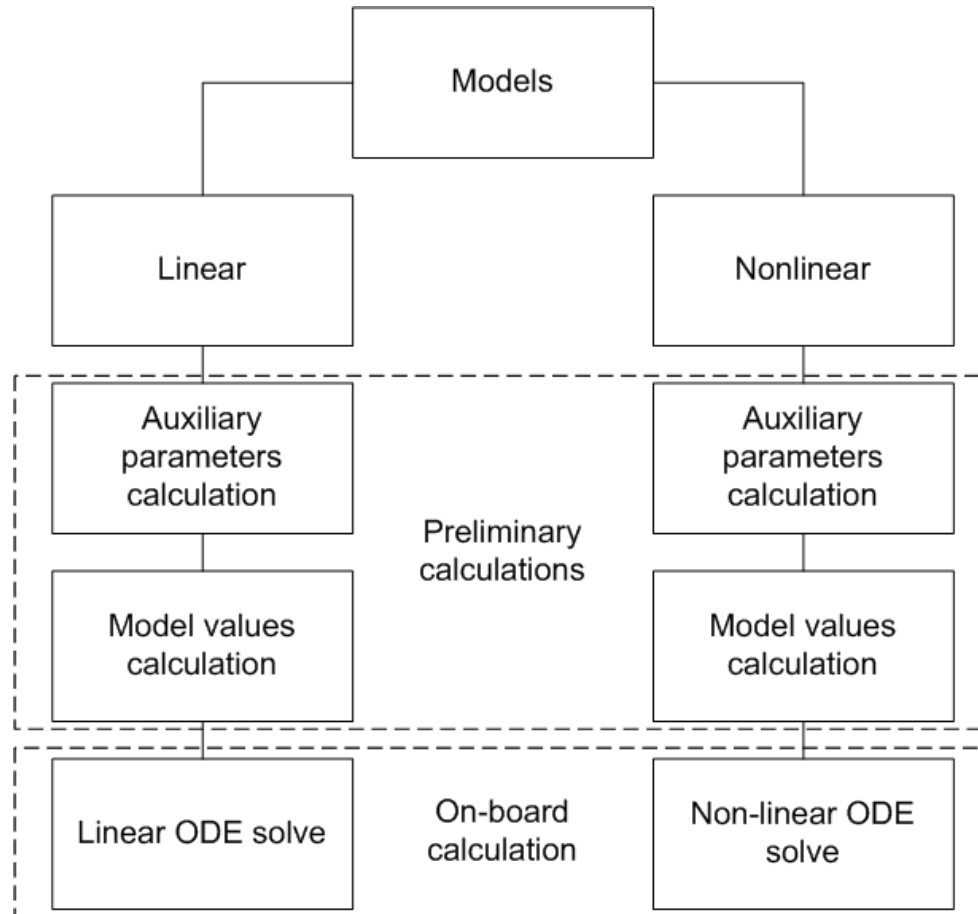
FEM calculation:

One position calculation: 40 minutes

Total time: about 120 minutes

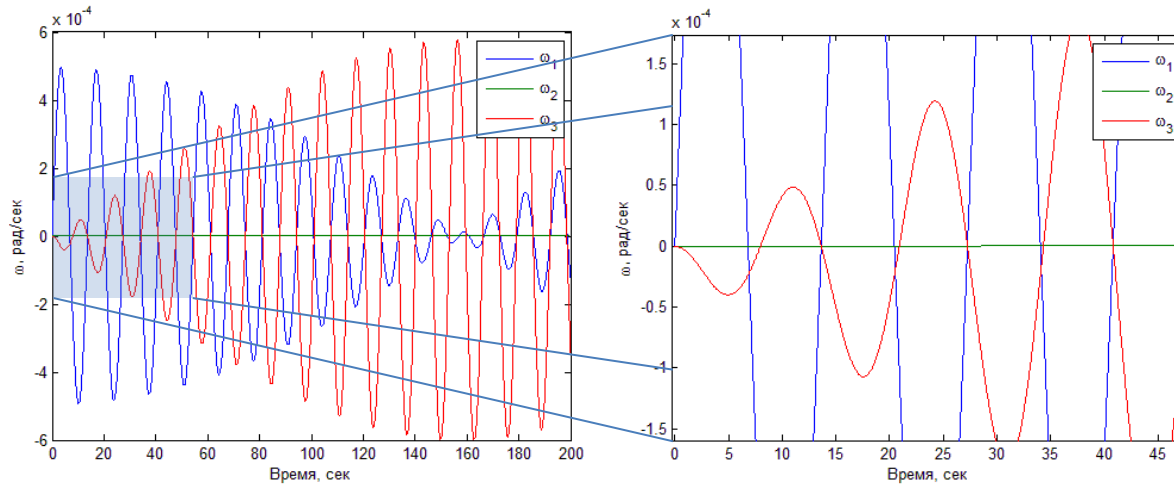


# Model on-board application

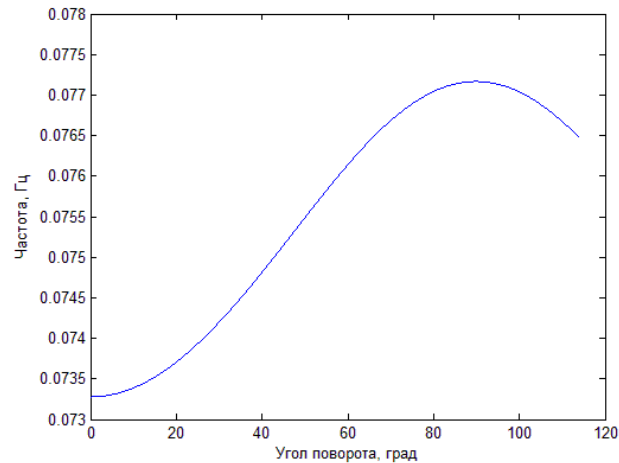


# Motion numerical modeling

## Angular velocity



## 1<sup>st</sup> mode frequency variation



# Conclusions

To describe multi-element angular dynamics it is possible to use normal modes for each element instead of those for the whole system. The approach decreases dramatically numerical modeling time, allows to model dynamics with hinged flexible element.

The work was supported by the RFBR No. **16-01-00634 A**